

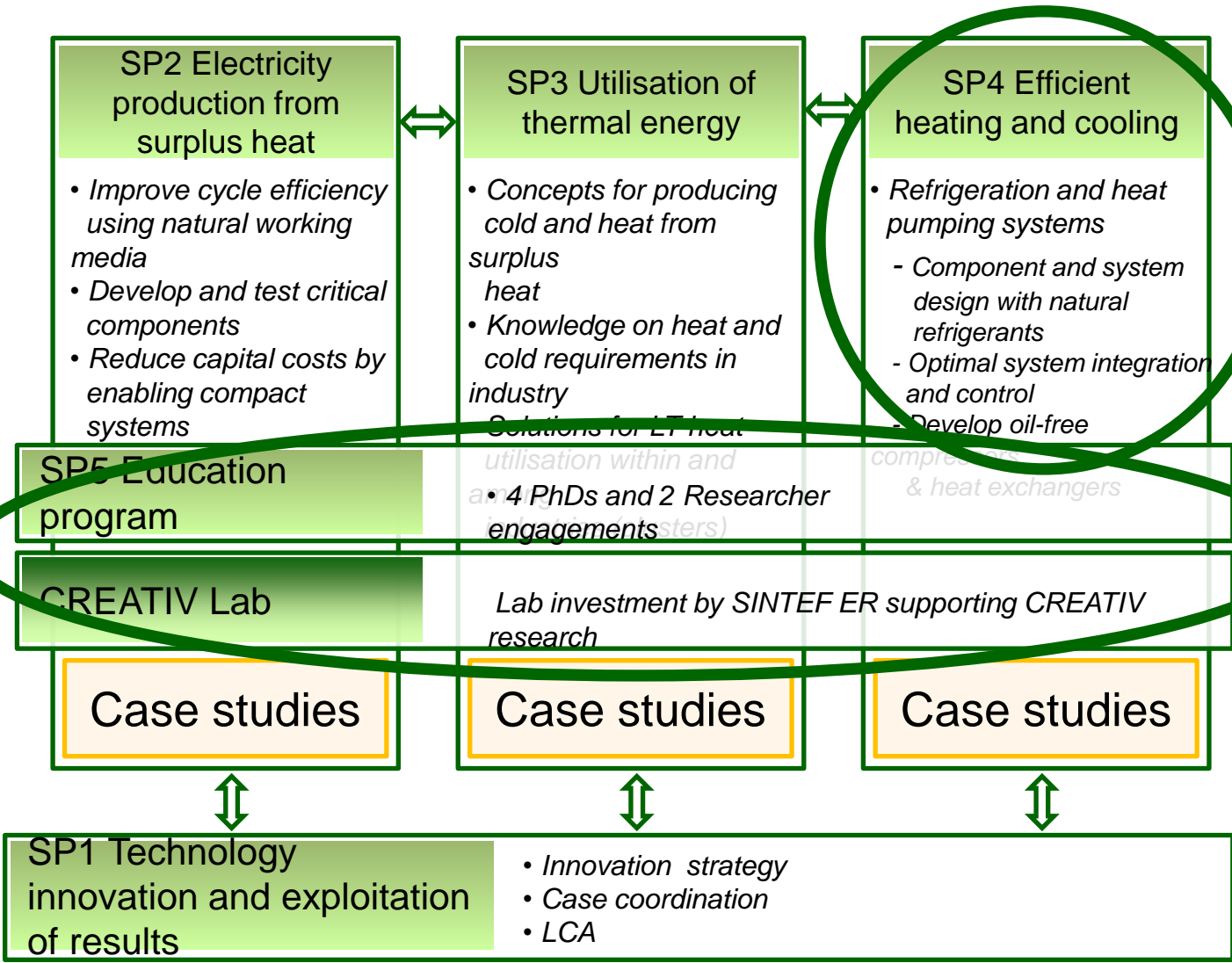
CREATIV

Refrigeration and heat pumping systems Some result examples

Competence project for Reduced Energy use through Advanced Technology
InnoVations

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Adjunct Professor, NTNU Energy and process eng.

130820 CREATIV Consortium Day



Development of more energy efficient and benign refrigeration technology

- The climate challenge is demanding environmentally benign **working fluids** (CO₂, NH₃, HCs and H₂O)
- Efficient **components** are crucial for energy efficiency
 - Heat exchangers, compressors, expanders, ejectors
- Development of improved **tools**, simulation models and test rigs, a necessary basis
- Development and implementation of improved **systems** for various application

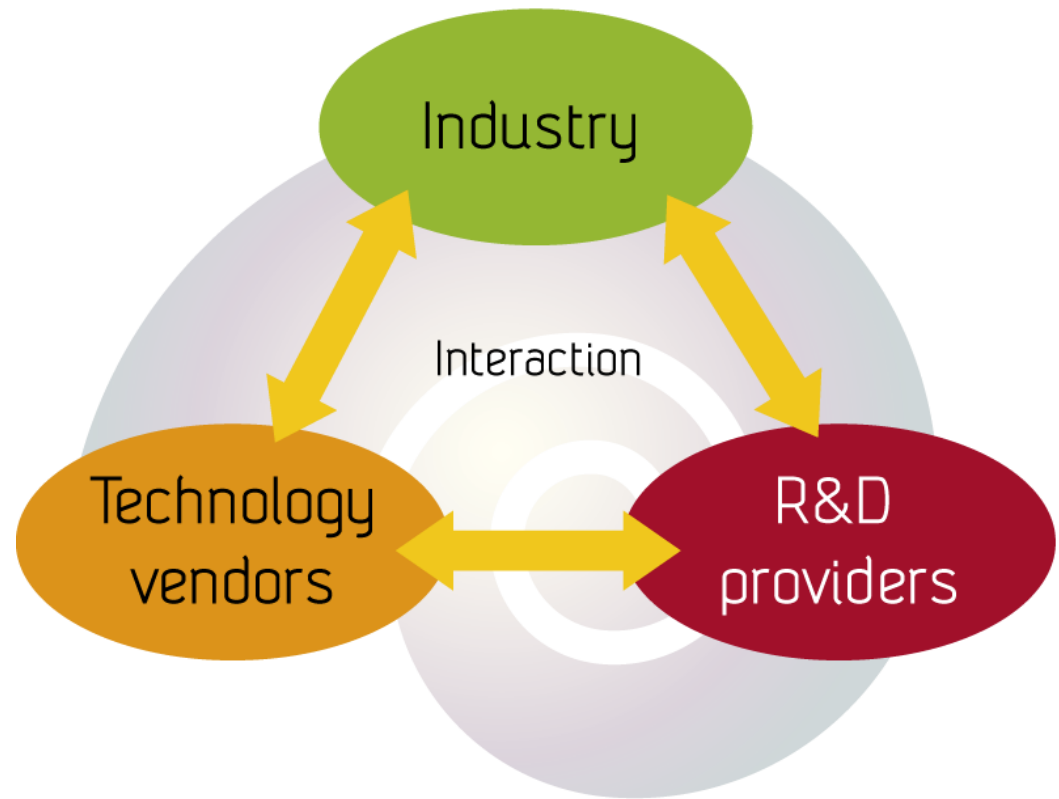
We need to reduce



emissions

Outline

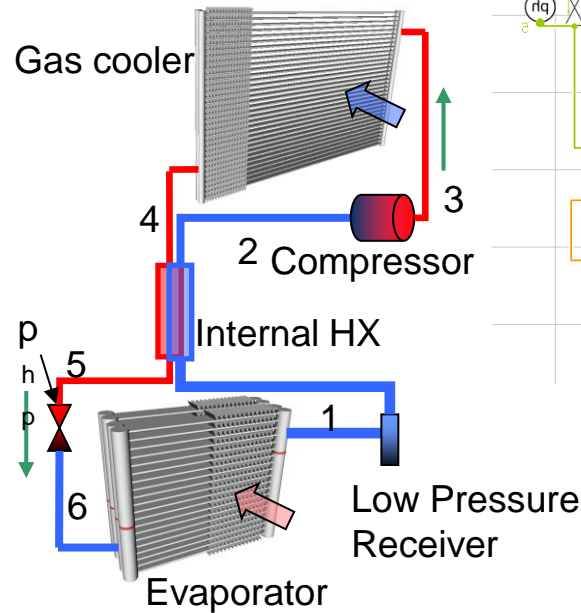
- Tool development
 - Modelling
 - Stationary and Dynamic
 - Component and system
 - Laboratory infrastructure
 - Compressor and hx test rig
- Component development
 - Compressors
 - Reciprocating
 - Turbo
 - Ejectors
 - Heat exchangers
- Applications
 - Supermarket
 - Space conditioning
 - RSW (refrigerated sea water)



Improved simulation models

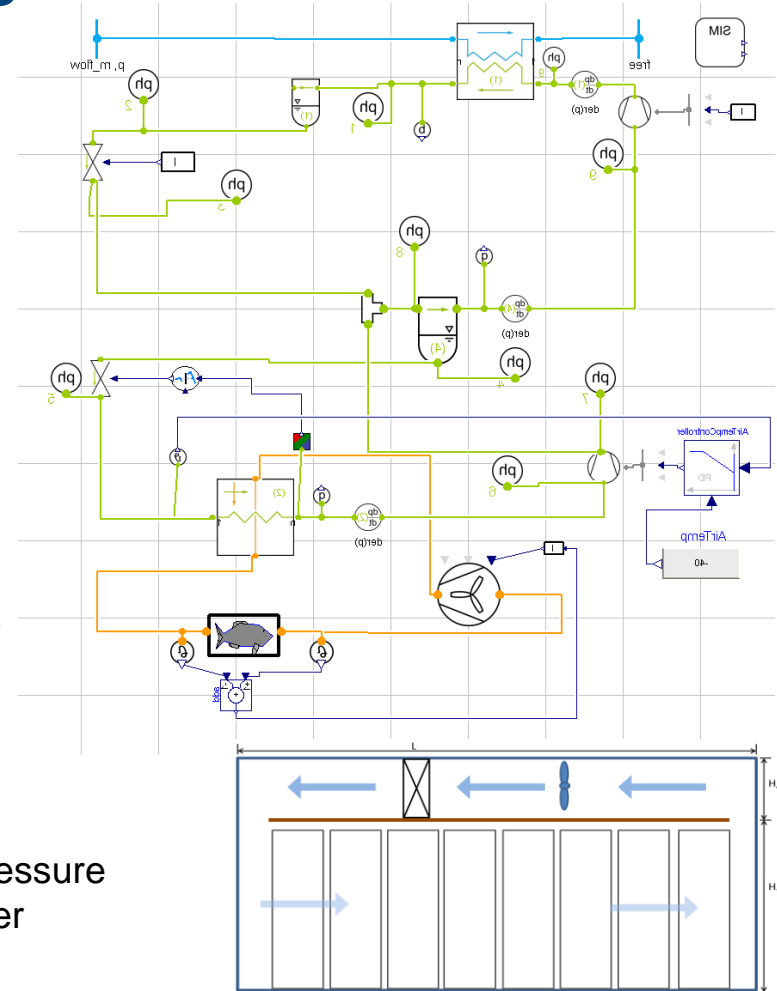
Enabling energy efficient design

- **Systems**
 - Refrigeration systems
 - Processes
 - Simulation and optimisation
- **Components**
 - Heat exch
 - Compressors
 - Ejectors
- **Stationary and dynamic**



System sim with detailed component models, stationary

Reference: Skaugen et al 2010

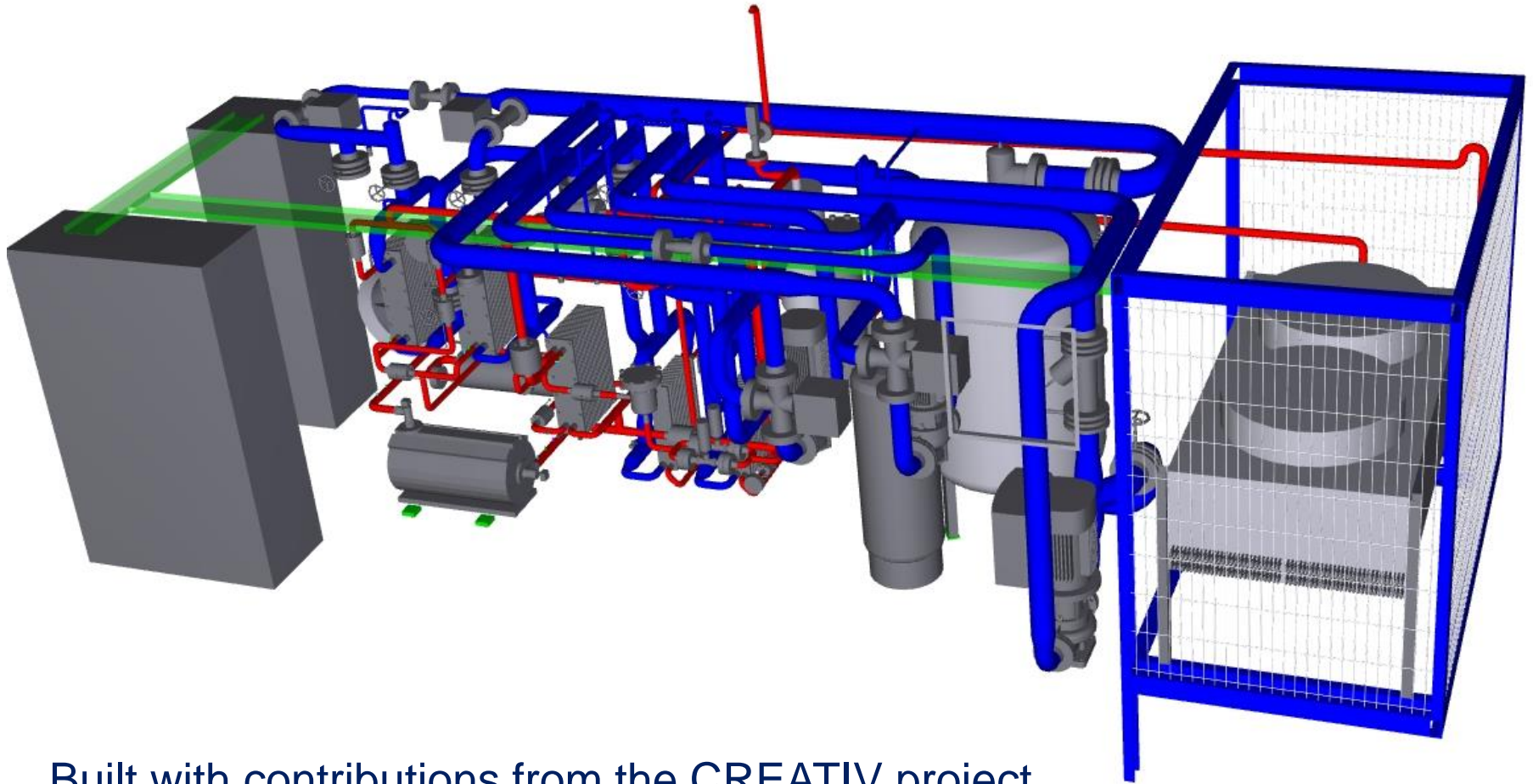


Dynamic model for freezing tunnel and refrigeration system

Reference: Andresen et al 2011

Improved laboratory infrastructure

R744 Compressor and Component Test Facility @ SINTEF

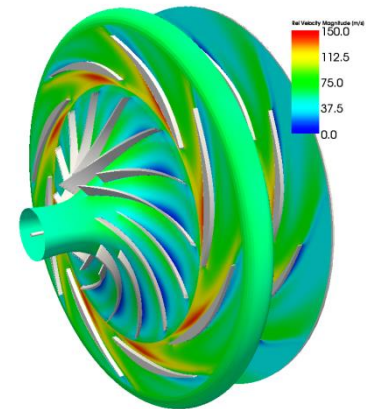


Built with contributions from the CREATIV project

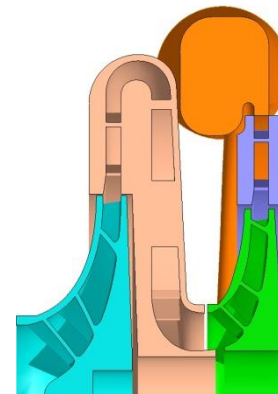
Component development

Efficient components are the building blocks

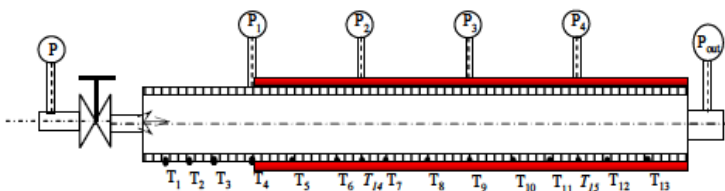
- **Compressors**
 - Reciprocating compr
 - Turbo compressor
 - Oil free technology
- **Ejectors**
- **Heat exchangers**
 - HP plate heat exch
 - Sublimator



CO₂ turbo compressor
Reference: Hafner et al 2011

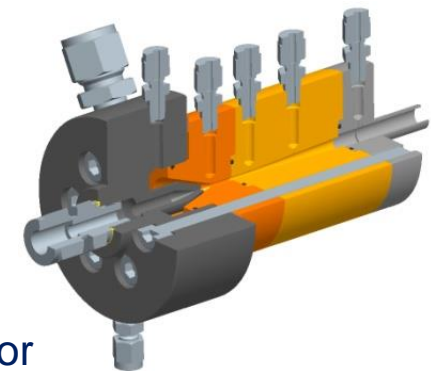
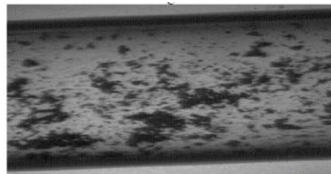


CO₂ plate hx, 140 bar
Reference: Kaori, r744.com



CO₂ SVE hx

Reference: Niu et al 2010



CO₂ ejector

Reference: Neksa et al 2010

Components:

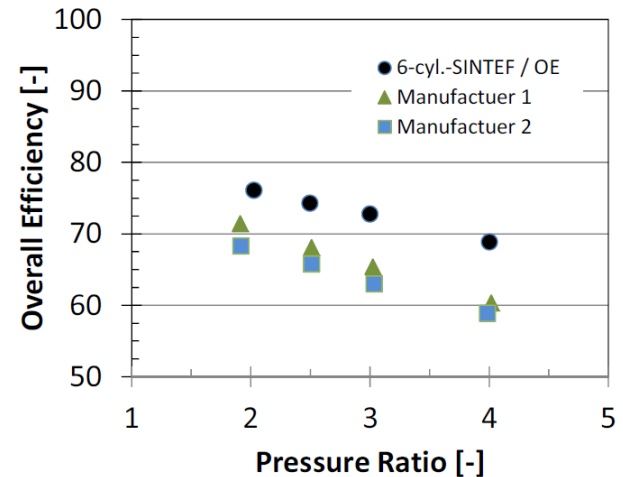
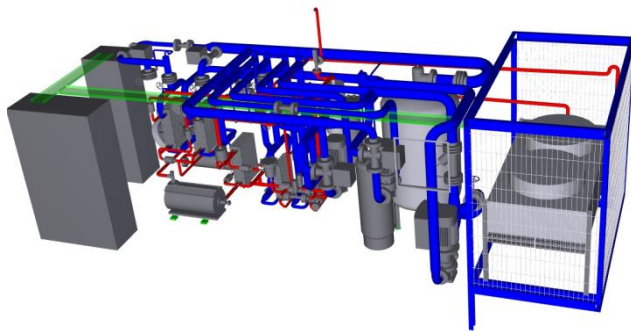
RECIPROCATING COMPRESSORS COOPERATION OBRIST E AND SINTEF

Glimpse

Compressor development

CO₂ compressor and test rig

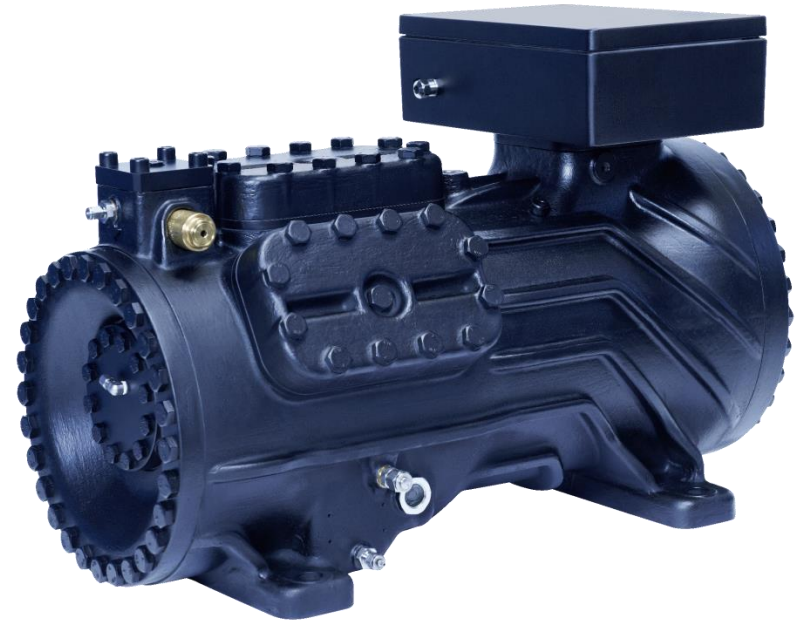
- **5-12 % improved energetic efficiency**



Joint development project between SINTEF and OE

18-90m³/h R744 compressor [2012]

Suitable for e.g. commercial refrigeration applications



technische Daten

Höhe x Breite x Länge [mm] **500 x 440 x 830**

Gewicht [kg] **286**

Volumenstrom [m³/h] **18 - 90**

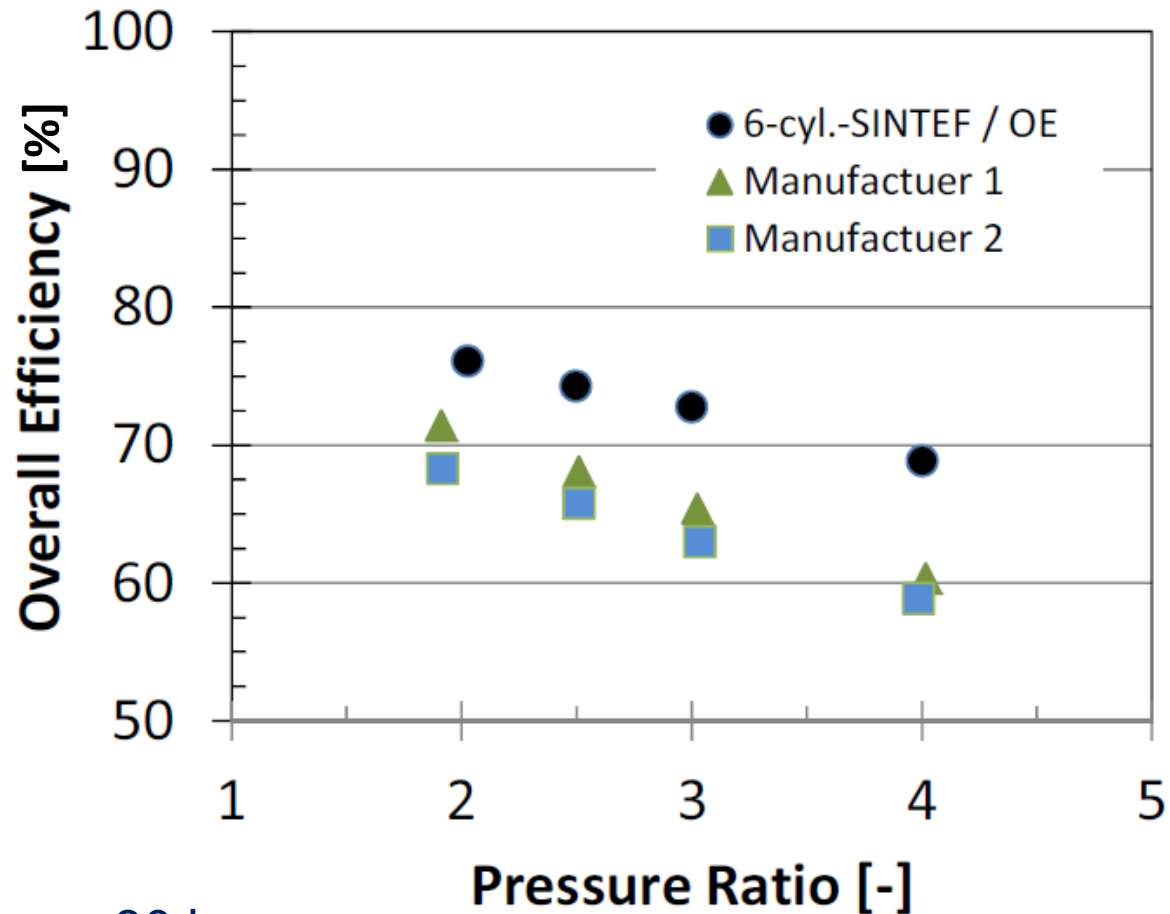
Hubraum [cm³] **380**

Max. el. Leistungsaufnahme [kW] **100**

Drehzahl [min⁻¹] **800 – 4 000**

Frequenzbereich [Hz] **53 - 267**

18-90m³/h R744 compressor [2012]



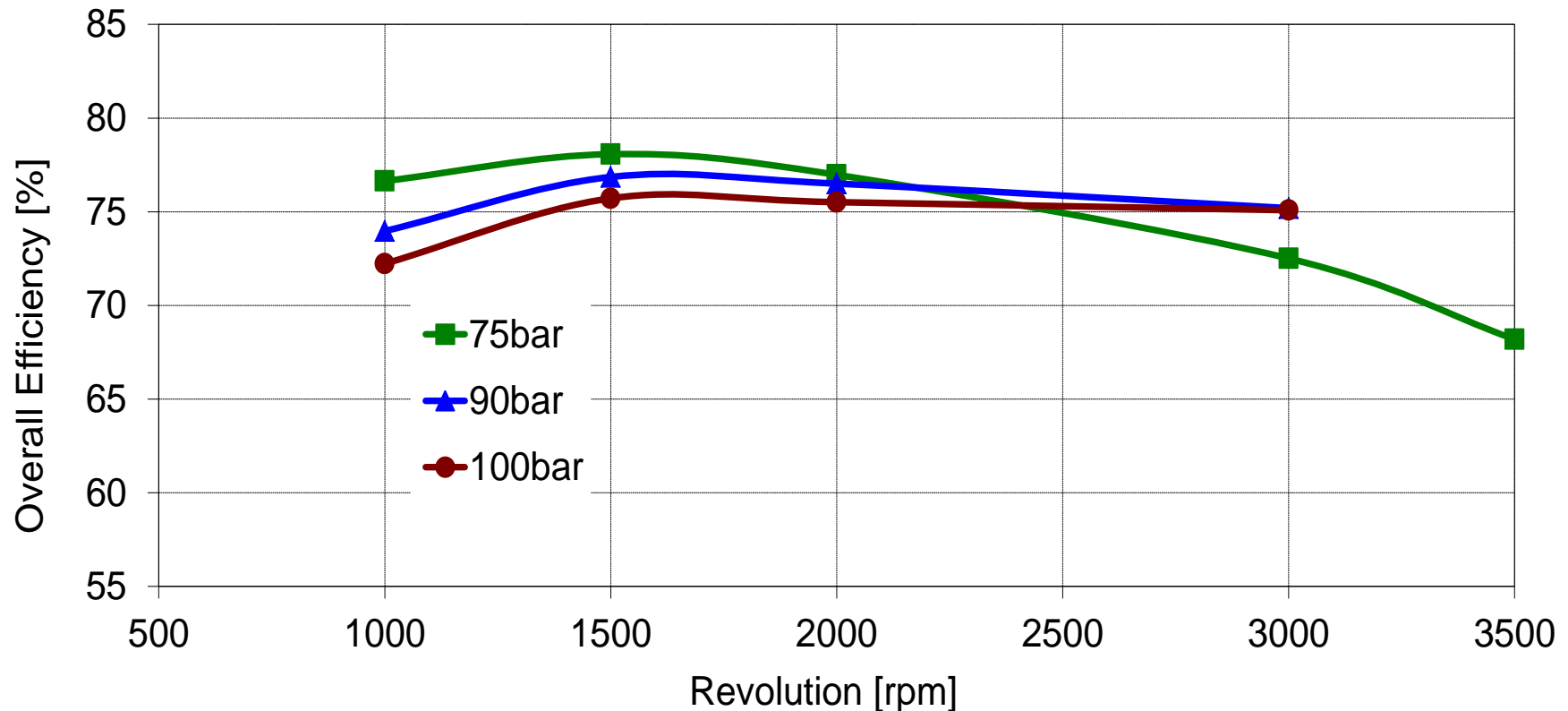
1500 rpm

discharge pressure: 80 bar

superheat at the compressor inlet: 10 K

18-90m³/h R744 compressor [2013]

Overall efficiency at varying speed, $p_0=35$ bara



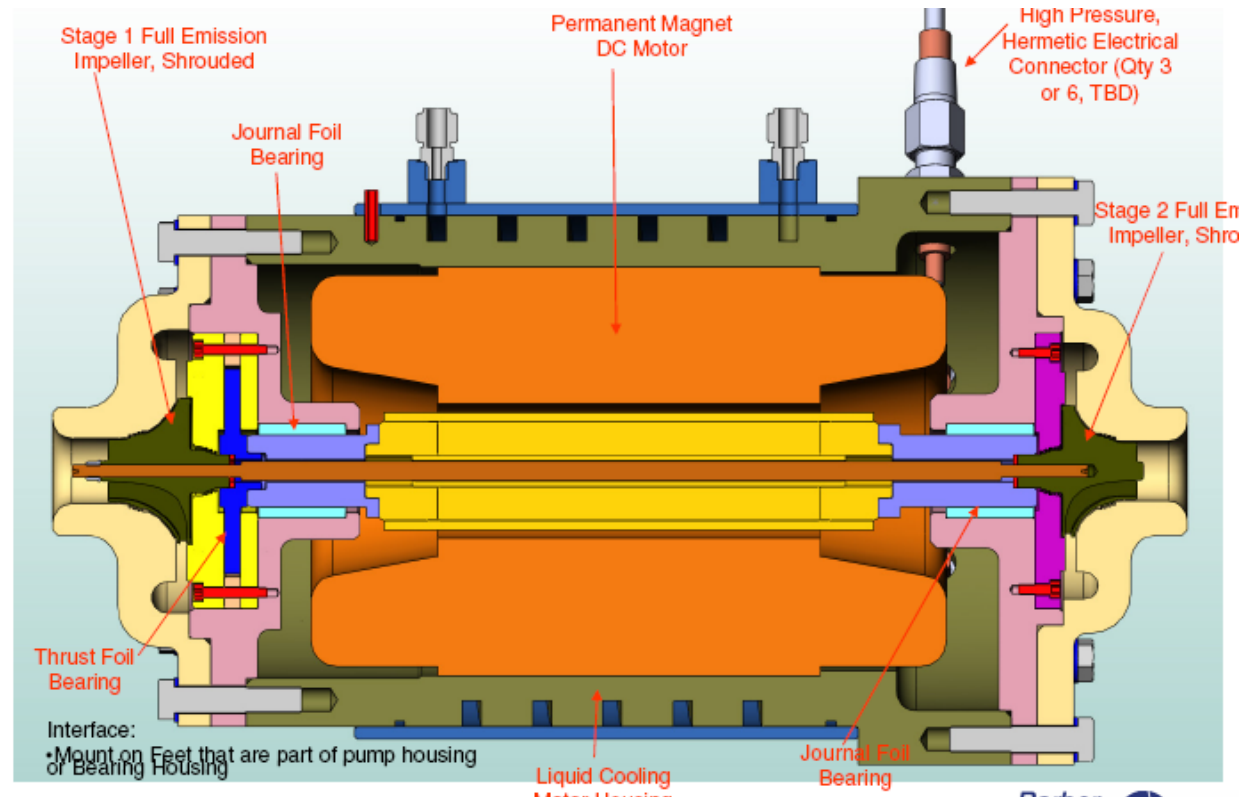
Components:

TURBO COMPRESSORS

PHD STUDENT BARTOZS KUS ET AL

Base case turbo concept for CO₂

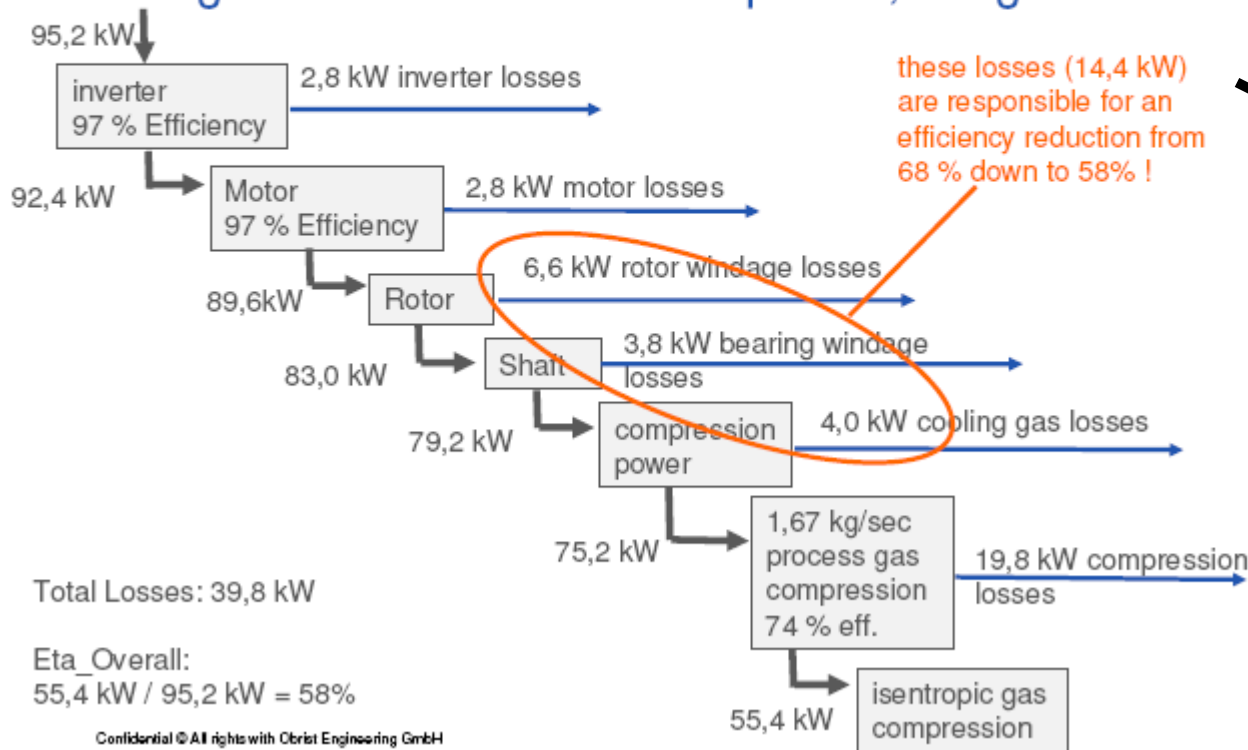
- Ship cooling
- Mass flow 1.6
- Pressure ratio: 2.13
- Pressure ratio **per stage: 1.46**
- Gas bearings
- **Aero efficiency: 75%**
- **Total efficiency: 60%**



Loss analysis base case



Design Point: 30/64bar - 47krpm - 1,67 kg/sec



What if we can get rid off these non-compression losses associated with high speed, but maintain 75% efficiency on the impeller side

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Efficiency curves

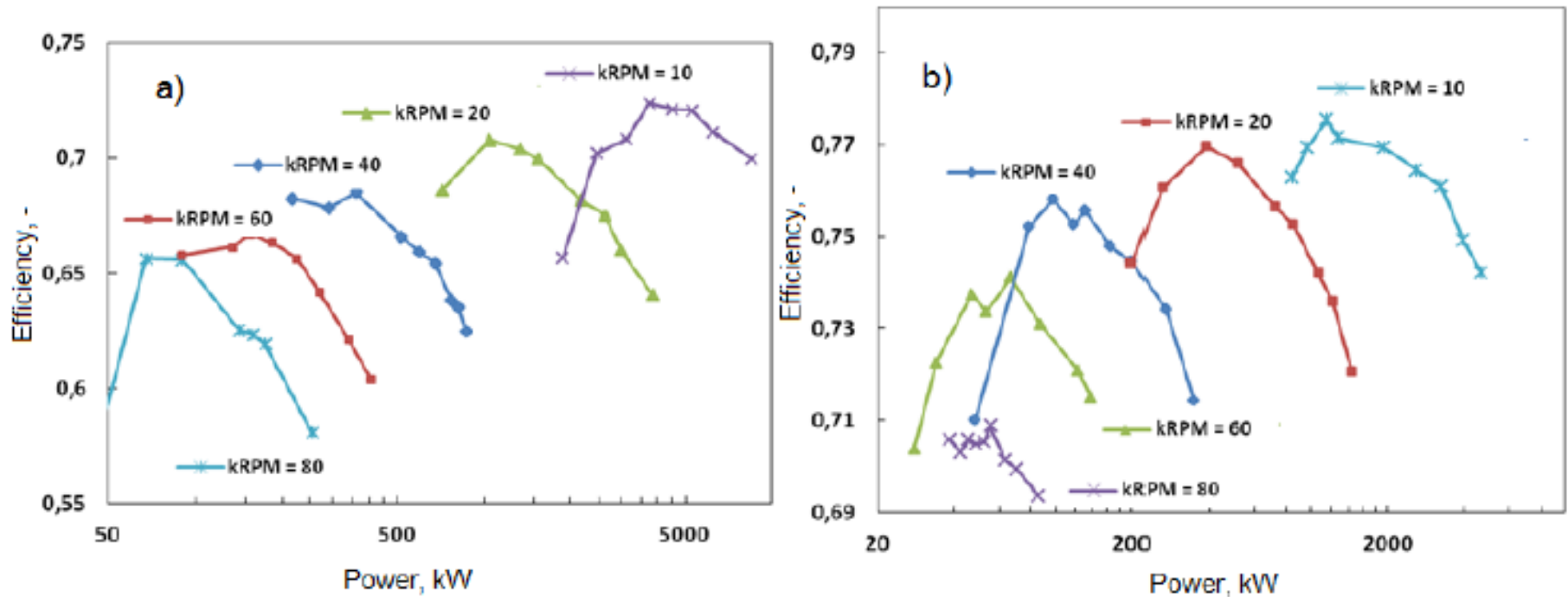
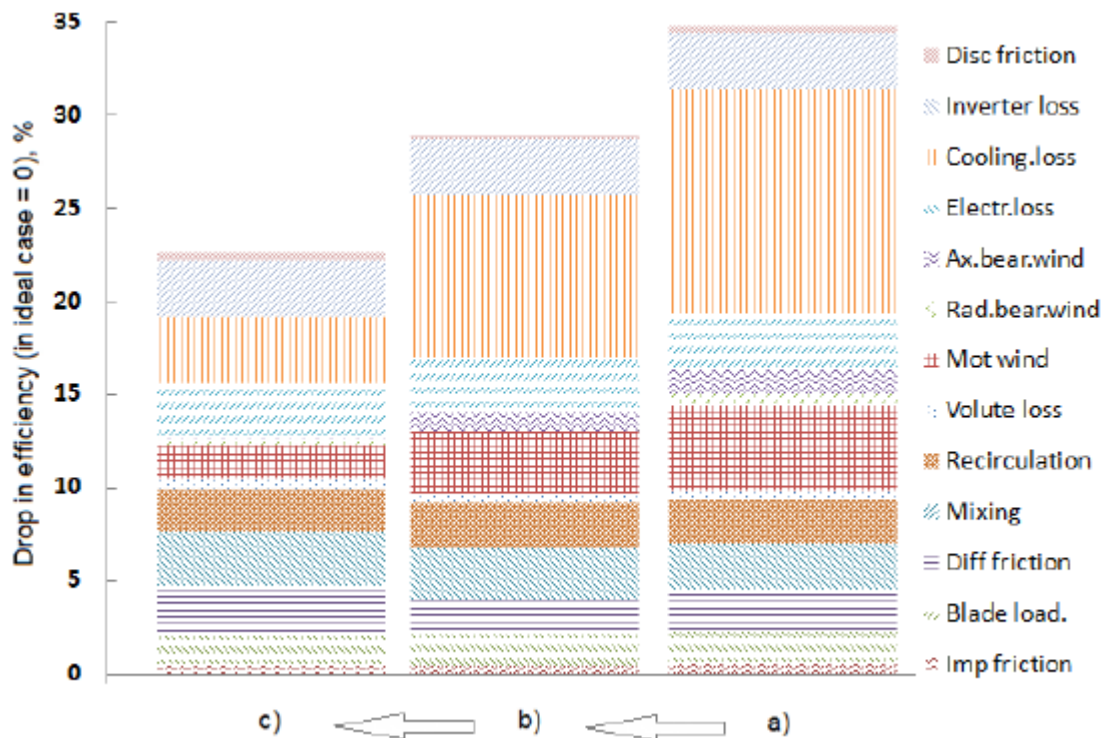


Figure 2. 2-stage (a) and 4-stage (b) compression efficiency for low stage loop in cascade refrigeration system.

Loss distribution



- a) Single 2-stage compressor, 80 krpm, 84 kW
- b) Single 2-stage compressor, 20krpm, 1333 kW
- c) Two 2-stage machines in series, 40 krpm, 98 kW in total



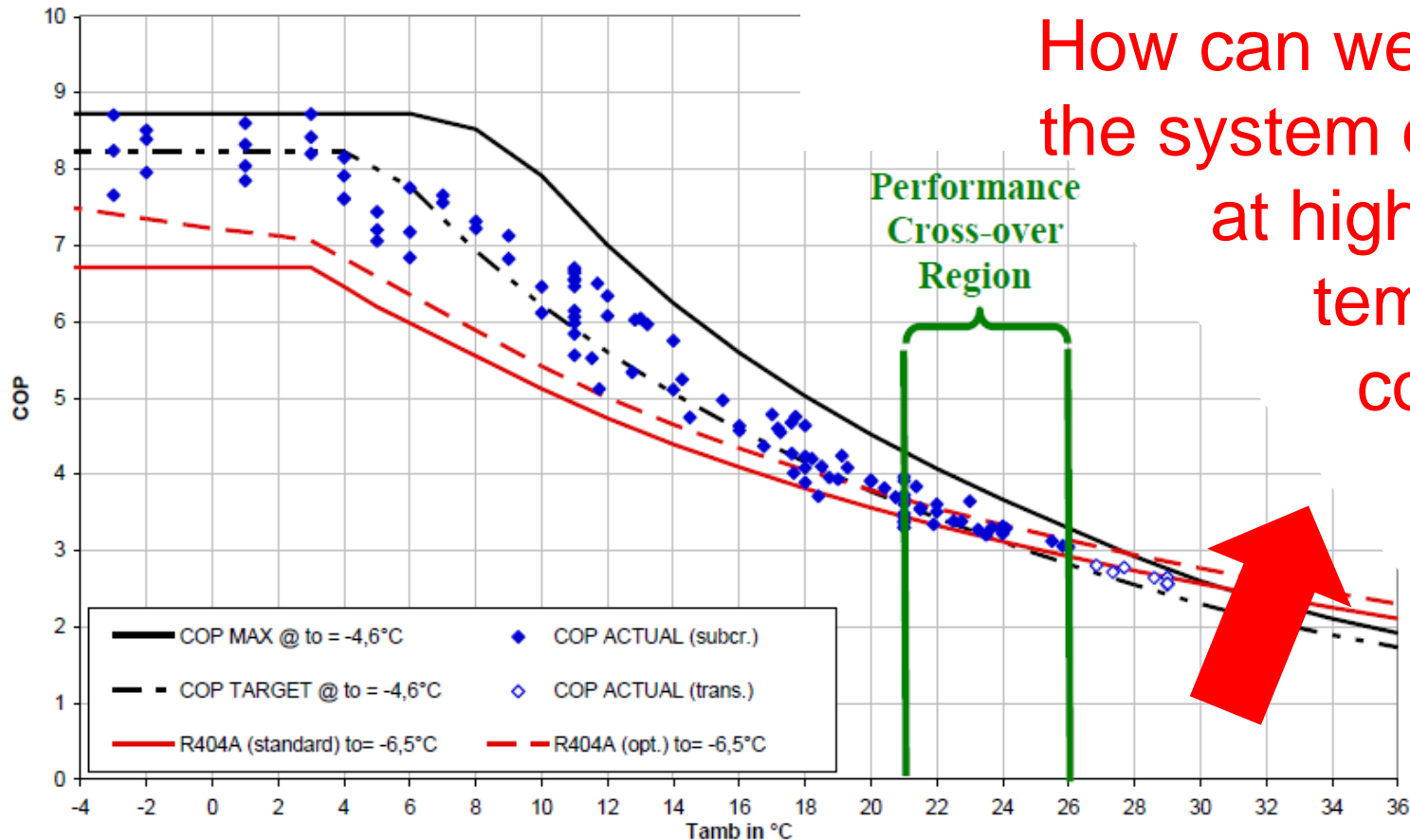
Components:

EJECTORS

Standard R744 Booster system for commercial refrigeration

Challenge:

How can we improve the system efficiency at high ambient temperature conditions?

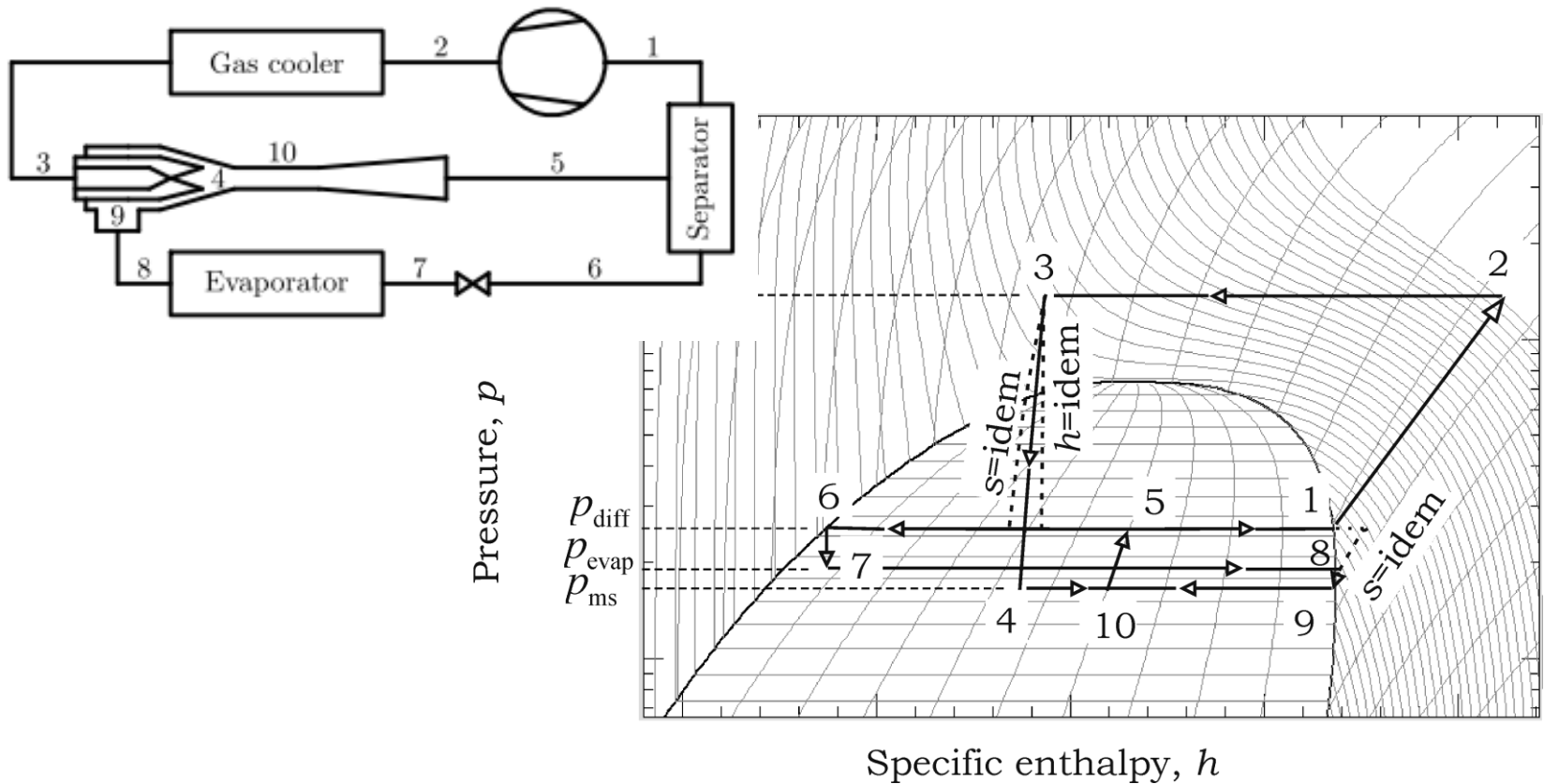


Ref. Carrier, ICR2011 Prag

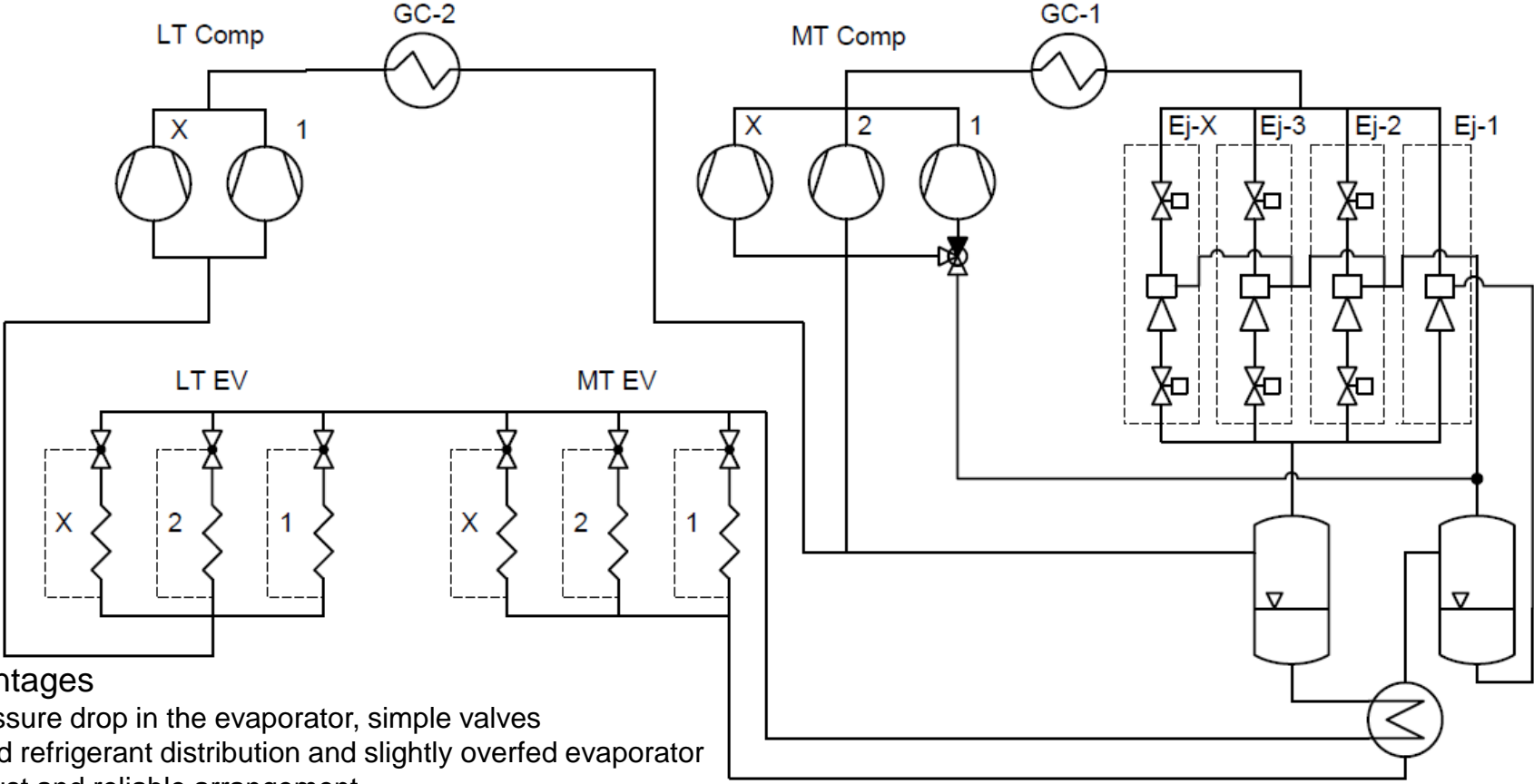
One possible way:

Transcritical R744 refrigeration cycle

Using a two-phase ejector for expansion work recovery and the corresponding pressure-specific enthalpy diagram



Proposal for next generation of R744 commercial refrigeration system



Advantages

- pressure drop in the evaporator, simple valves
- good refrigerant distribution and slightly overfed evaporator
- robust and reliable arrangement
- lubricant management in separator

© SINTEF 2012

Components:

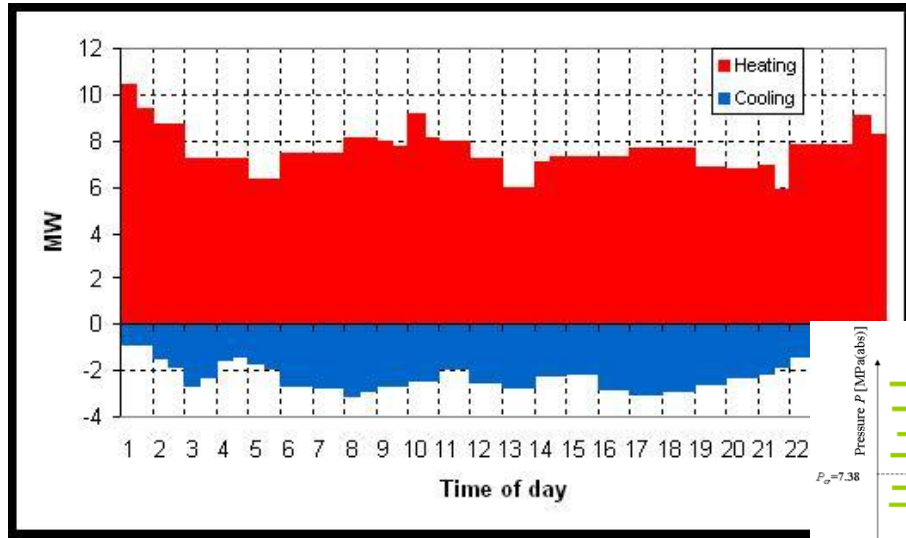
SOLID CO₂ SUBLIMATOR

NIU ET AL

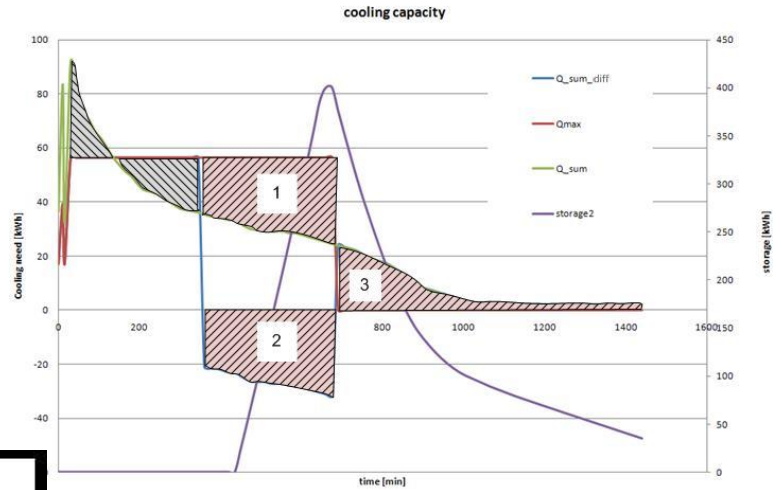
Energy efficiency

Heat and cold accumulation

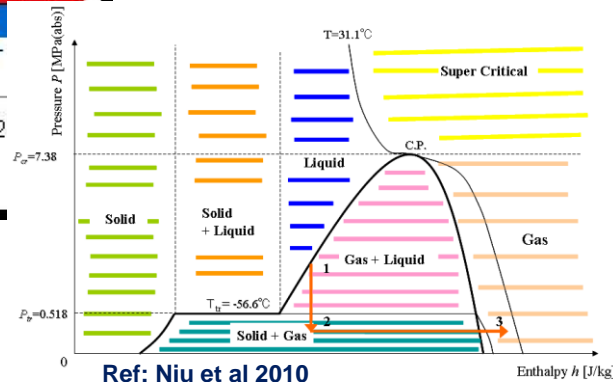
- CO₂ for low temperature cold accumulation



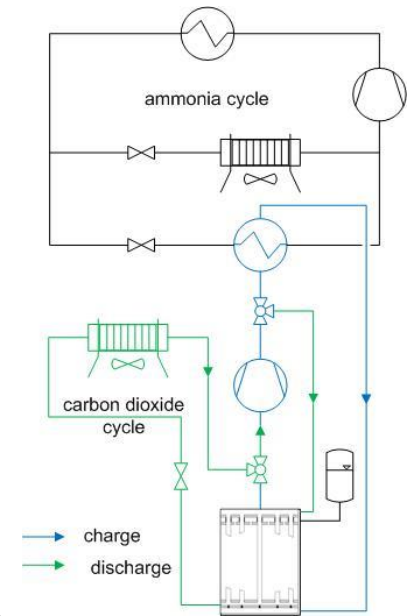
Ref: Nordtvedt et al 2011



Ref: Hafner et al 2011

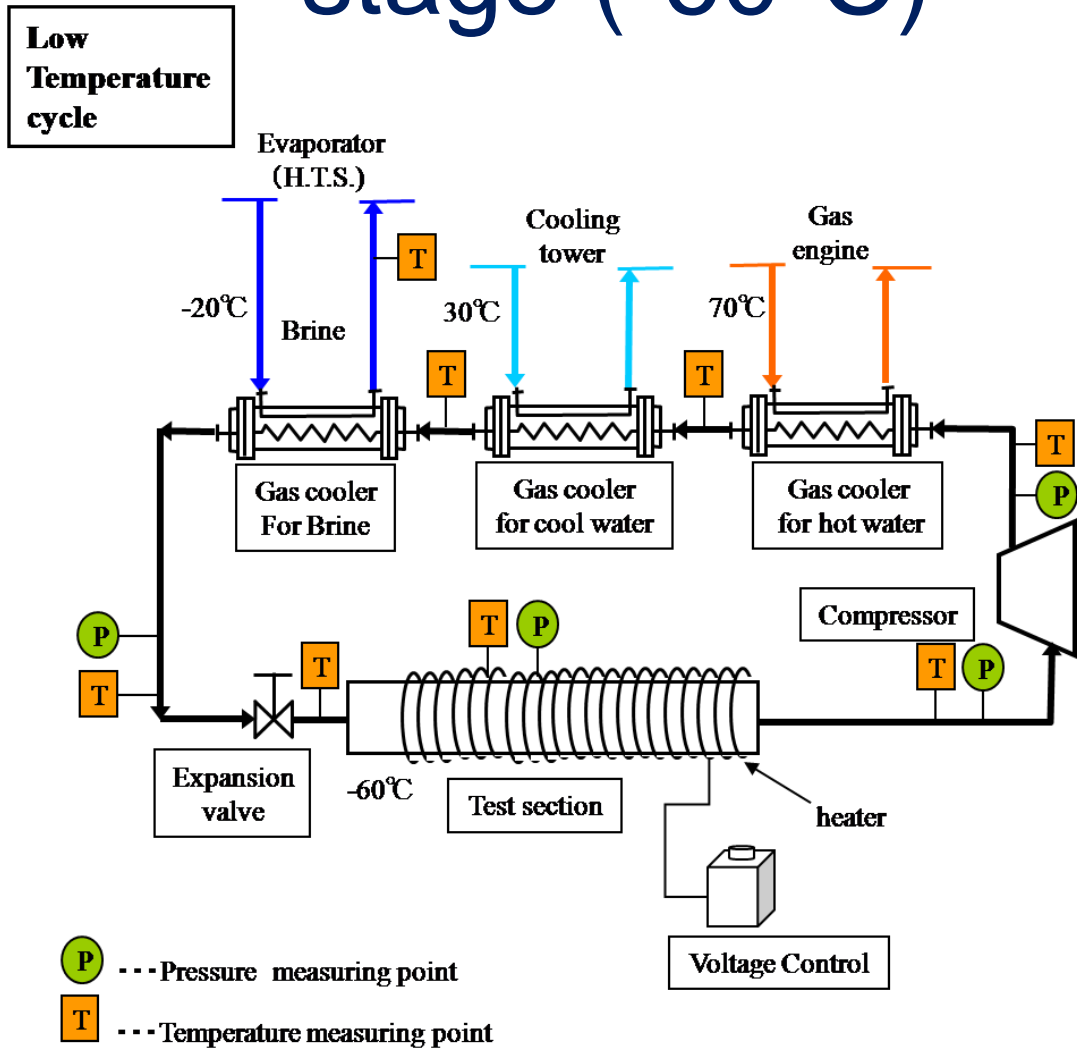


Ref: Niu et al 2010

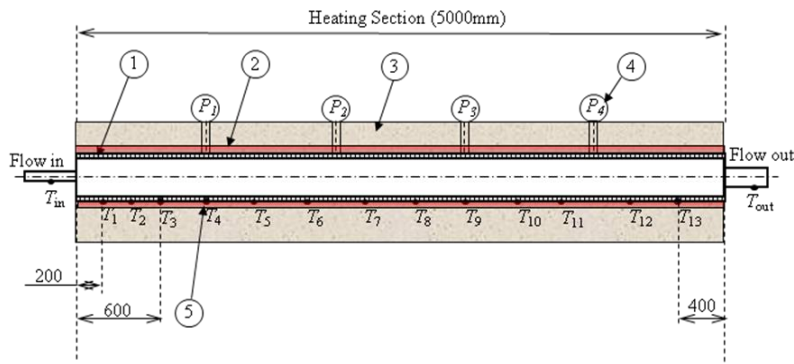
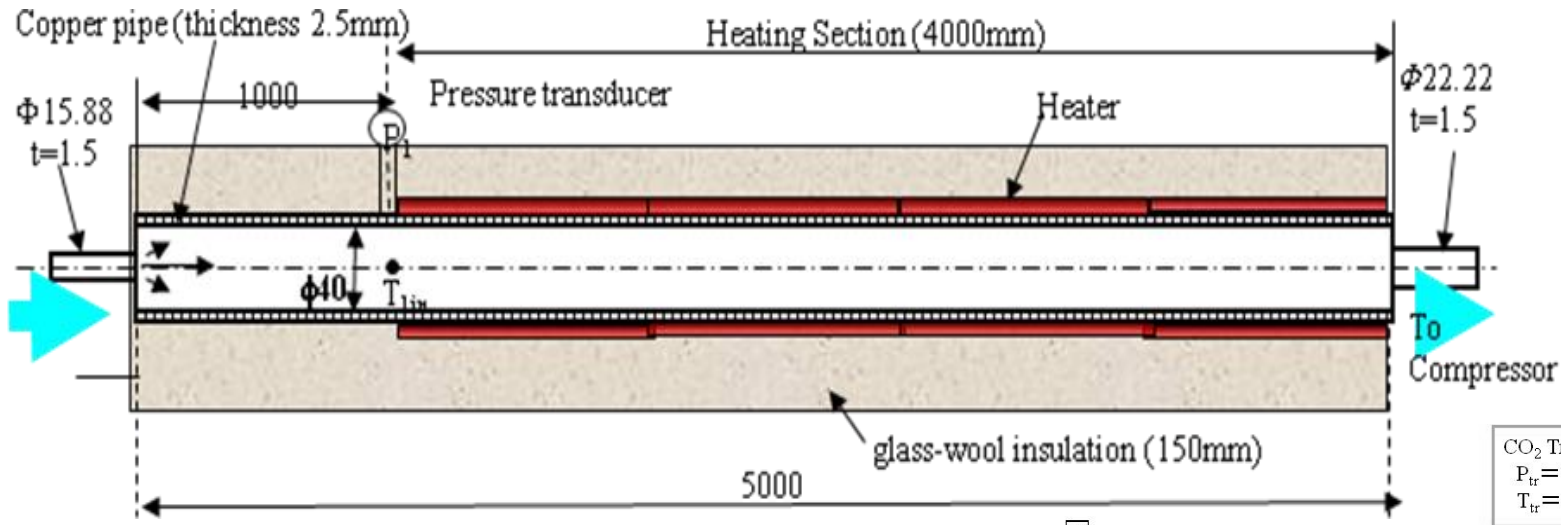


→ charge
→ discharge

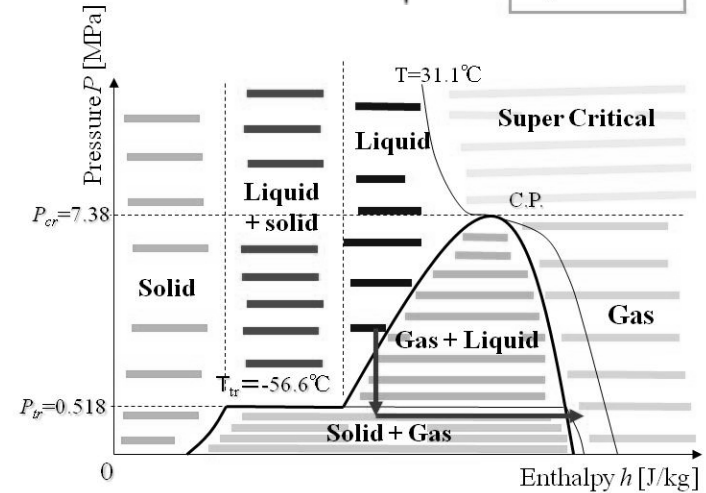
Schematic of low temperature stage (-60°C)



Test section and operation

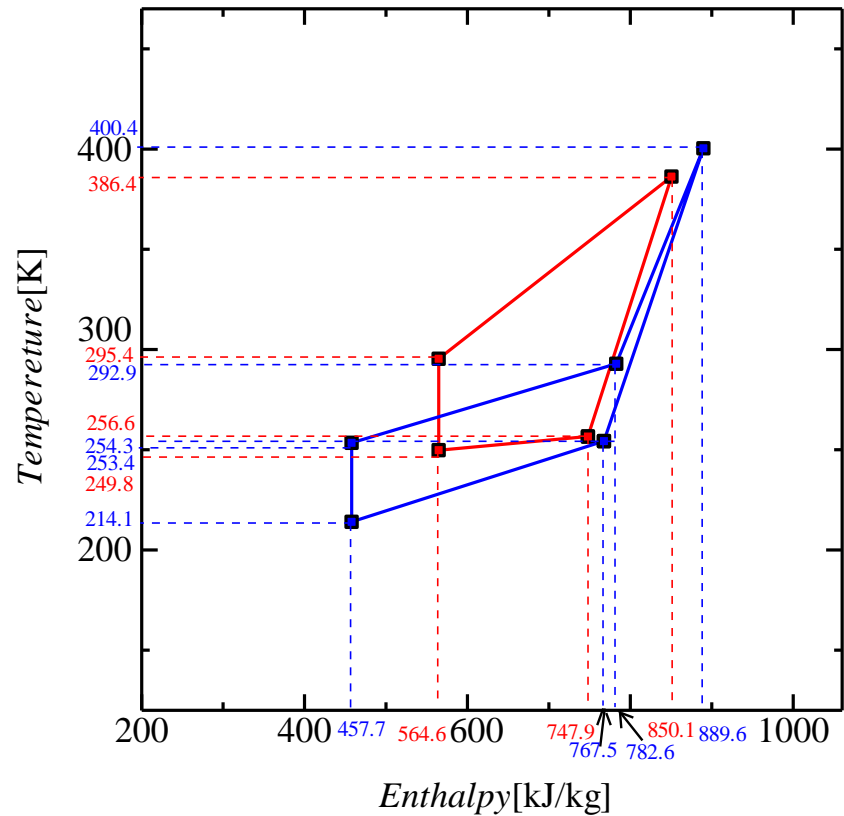
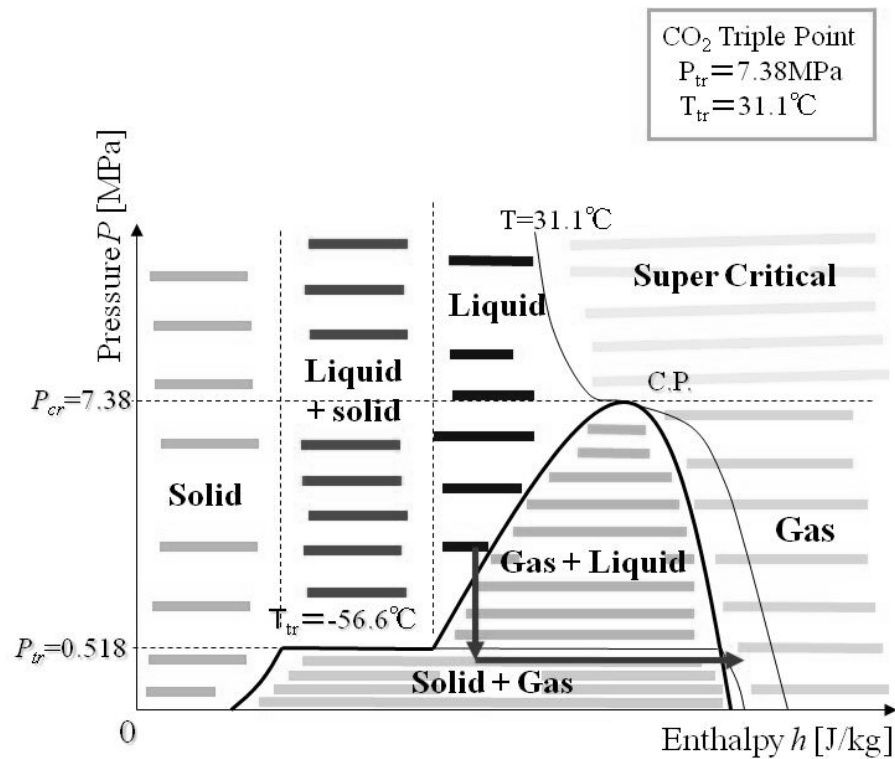


- ① Copper pipe ($d=40\text{mm}$, $t=2.5\text{mm}$, $L=5000\text{mm}$)
- ② Heater (silicon rubber)
- ③ Glass-wool insulation (150mm)
- ④ Pressure transducer (4 points, uniformly distributing along test section)
- ⑤ Thermocouple (15 points)

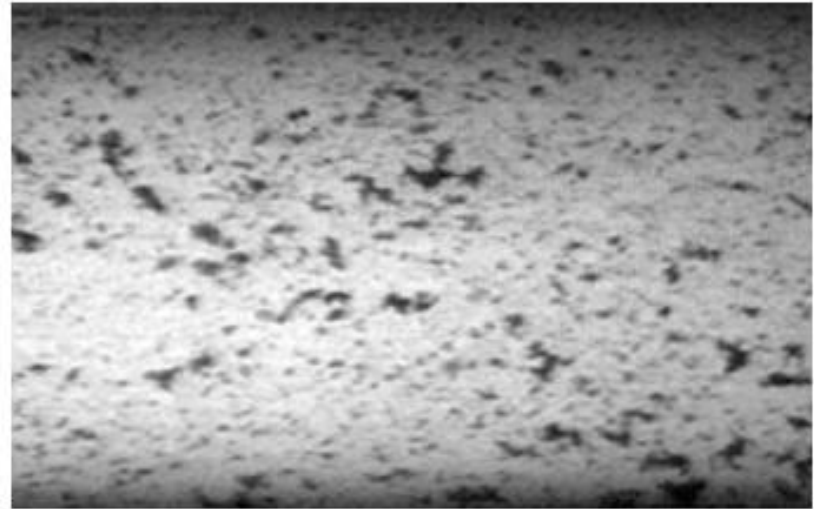


Throttling from liquid to solid-vapour state
Sublimation of CO₂ dry ice to gas

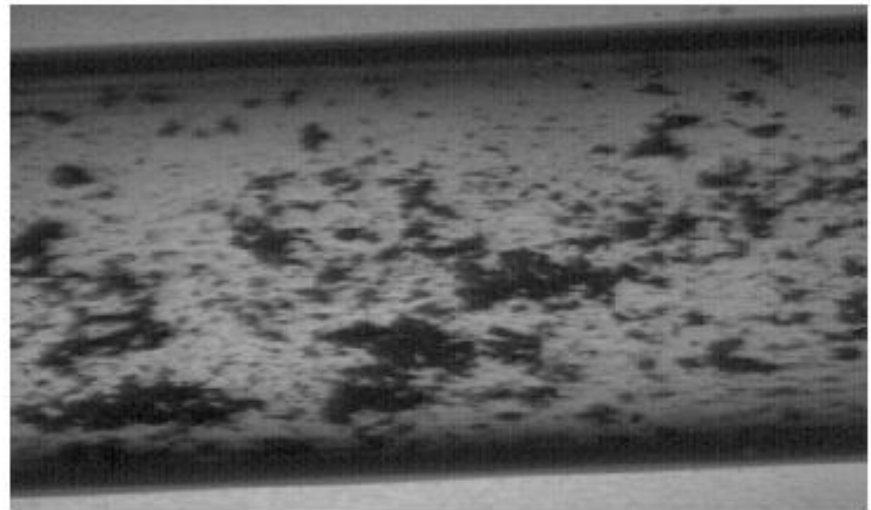
Cycle operation



Visualisation



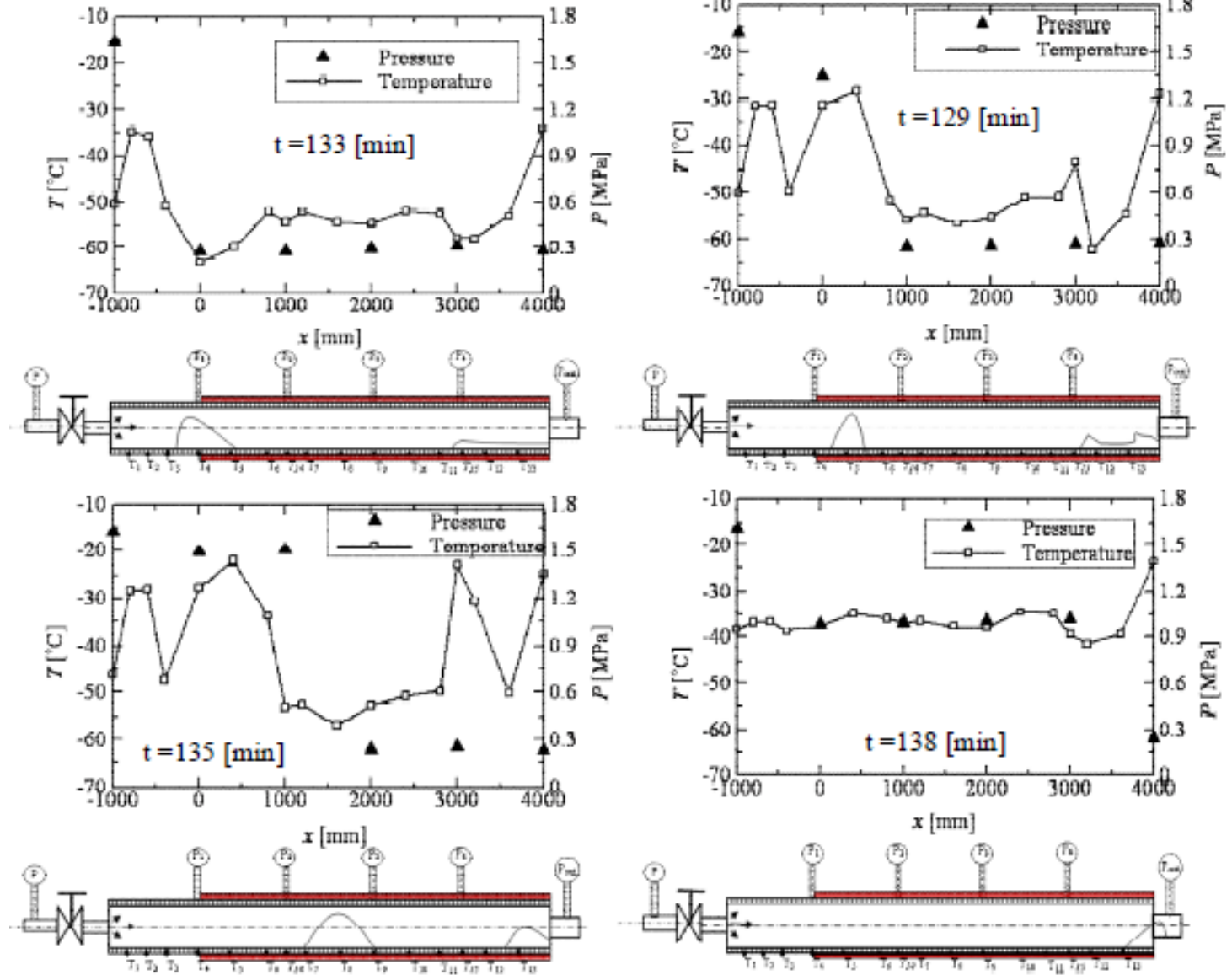
(a) CO₂ solid-gas flow at opening conditions of 15mm of expansion valve



(b) CO₂ solid-gas flow at at opening conditions of 10mm of expansion valve

Sedimentation

Variations of local pressure and wall temperature with the dry ice sedimentation inside the test section at different times



Case:

SUPERMARKET ENERGY SYSTEMS

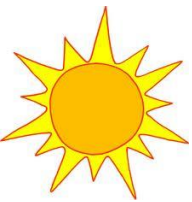
***PRESENTED BY ARMIN HAFNER IN THE
NEXT PRESENTATION***

Case:

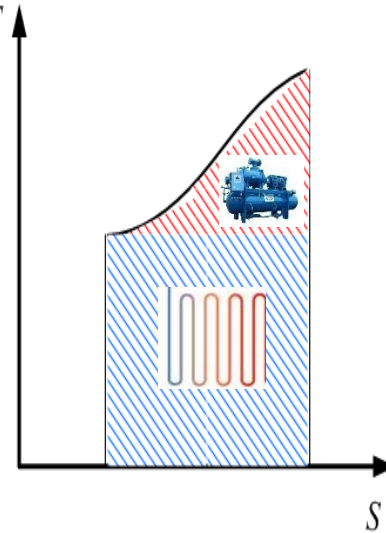
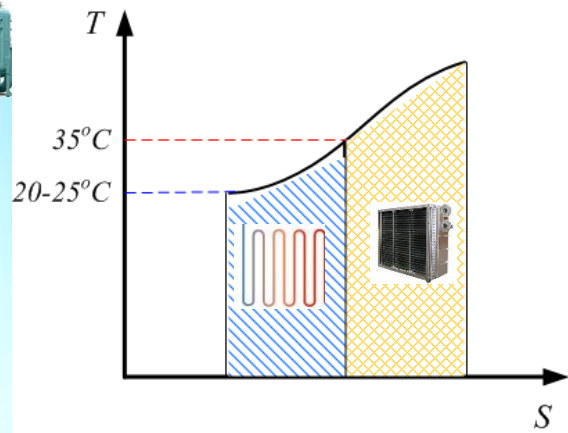
HEAT PUMPS FOR SPACE CONDITIONING

POSTDOC HAITAO HU ET AL

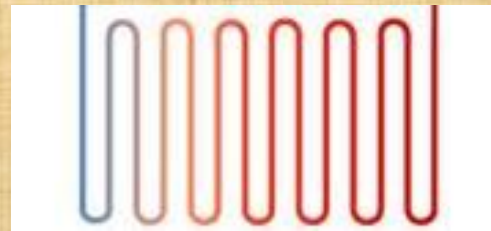
1. Motivation



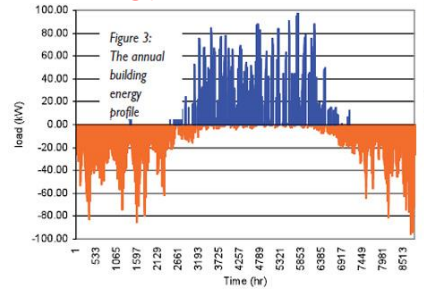
R744 Air conditioning system



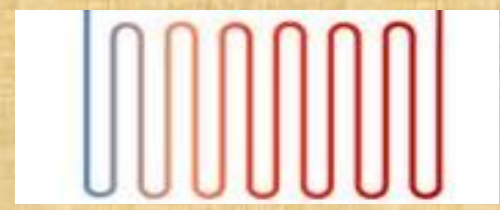
GSHX



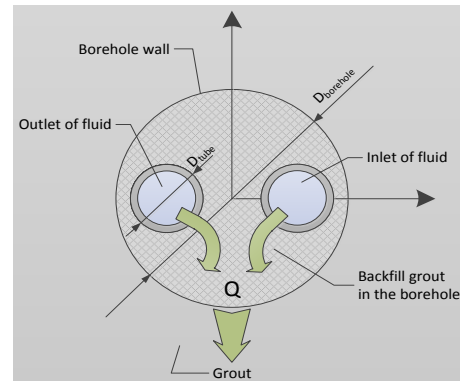
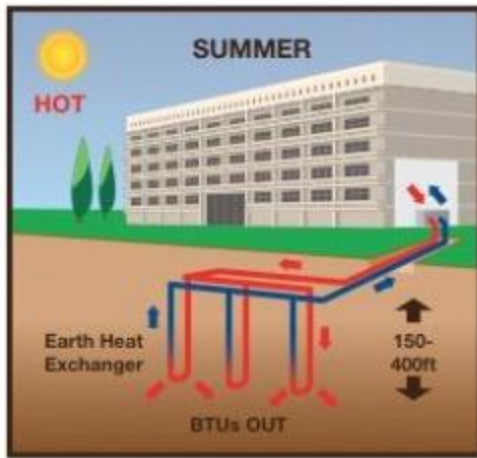
Energy load balance



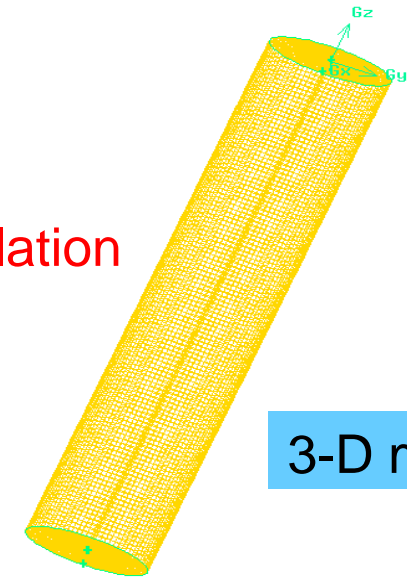
GSHX



2. CFD simulation on performance of GSHX



CFD simulation



3-D model

Influence factors

HX length (m);

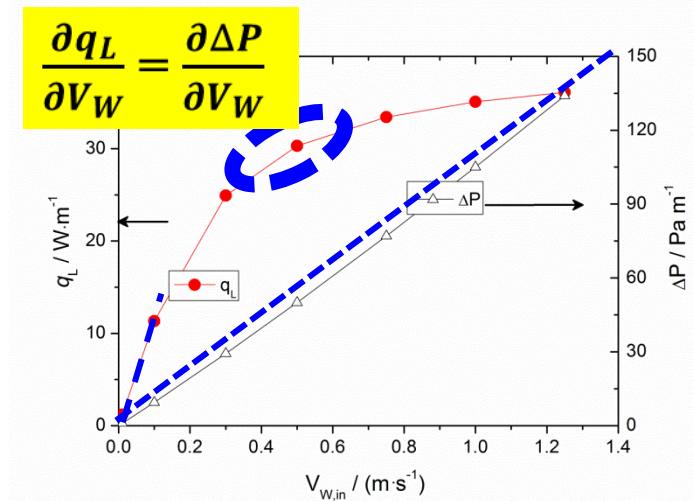
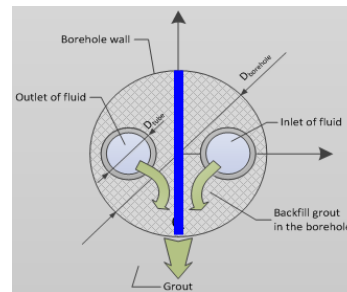
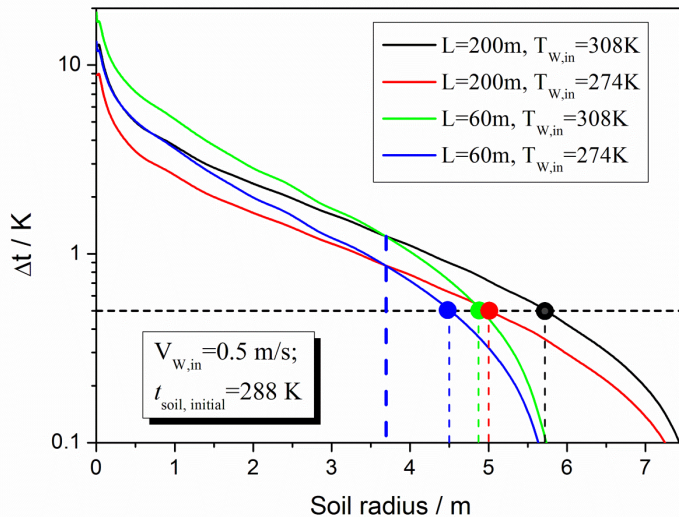
Fluid inlet temperature (K); Fluid inlet velocity (m/s)

Initial soil temperature (K); Soil types for different areas

Backfill material (Water, air, different soils)

3. CFD simulation on performance of GSHX (Results)

- ✓ Distance between HXs should be 9~12 m;
- ✓ $q_{200} \approx 0.9 q_{60}$ (W/m); $Q_{200} \approx 3.0 Q_{60}$ (W);
- ✓ Adding insulation board enhances heat flux by 10%;
- ✓ Optimal backfill material ($\alpha=2.0\sim 3.0$ E-6 m²/s);
- ✓ The optimal fluid velocity is 0.5~0.7 m/s.



Systems:

CO₂ RSW (REFRIGERATED SEA WATER)
LADAM ET AL (2012)

Glimpse

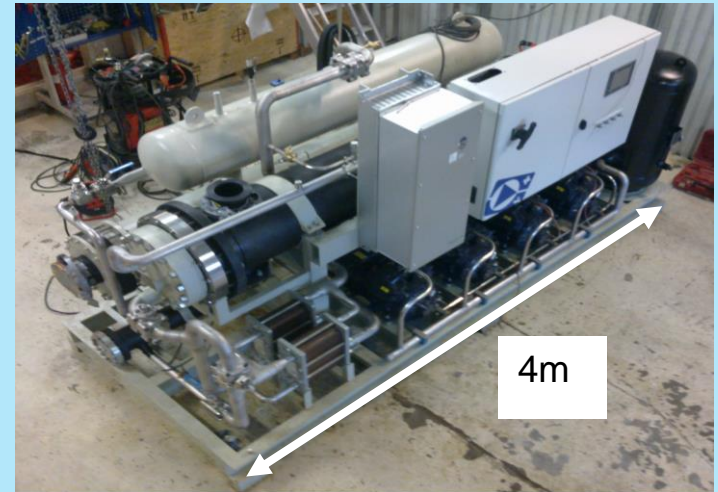
Fish industry

CO₂-RSW prototyp er installert på fiskebåt

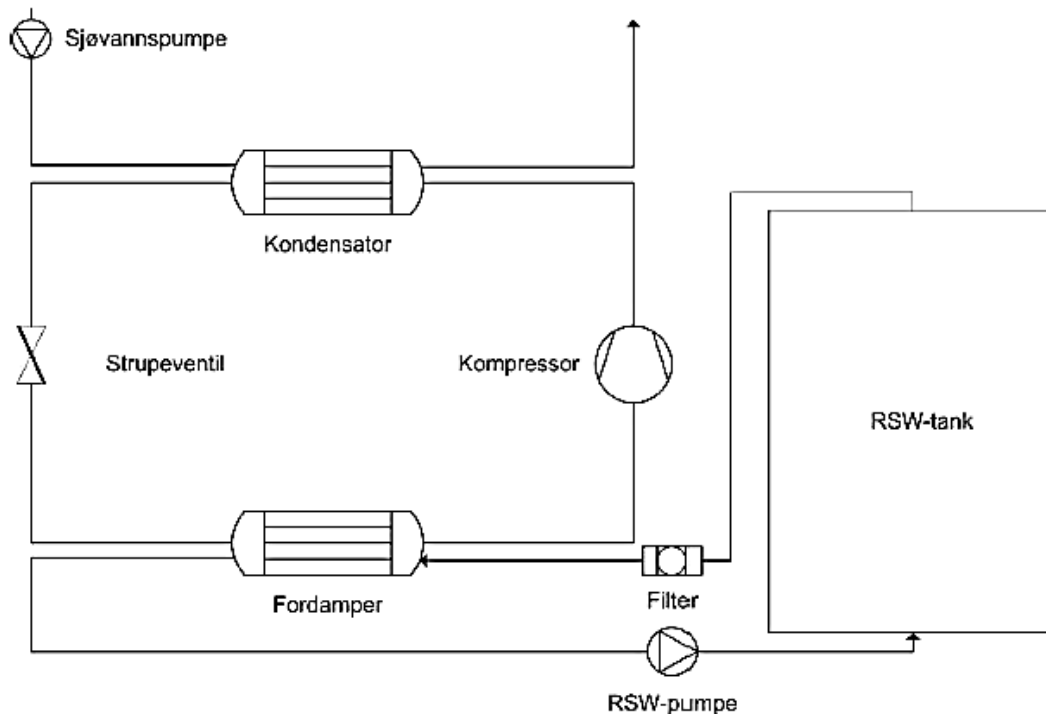
Miljøvennlig, ikke giftig kuldemedium, kompakt

Kan gi **40 % redusert energibruk**

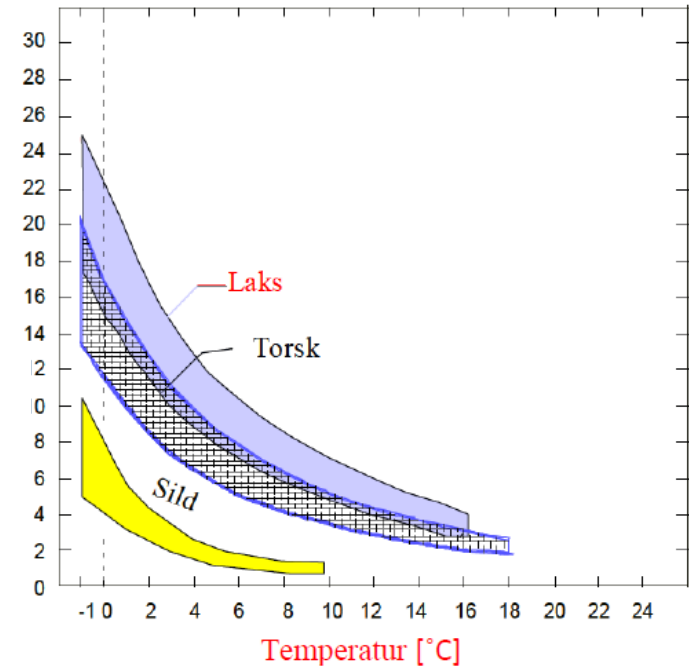
Egnet for retrofit



RSW



ildbarhetsdøgn



EU directive 92/48/EEC:

- $< 3^{\circ}\text{C}$ within 6 hours
- $< 0^{\circ}\text{C}$ within 16 hours

CO₂ RSW for fishing vessels: **impossible mission!**

- R22 phaseout 2015
- HFC not env alternative
- NH₃ not always suitable



CO₂ RSW, operating experience 250kW plant onboard Båragutt MS

Design and analysis

End customer



Financing design

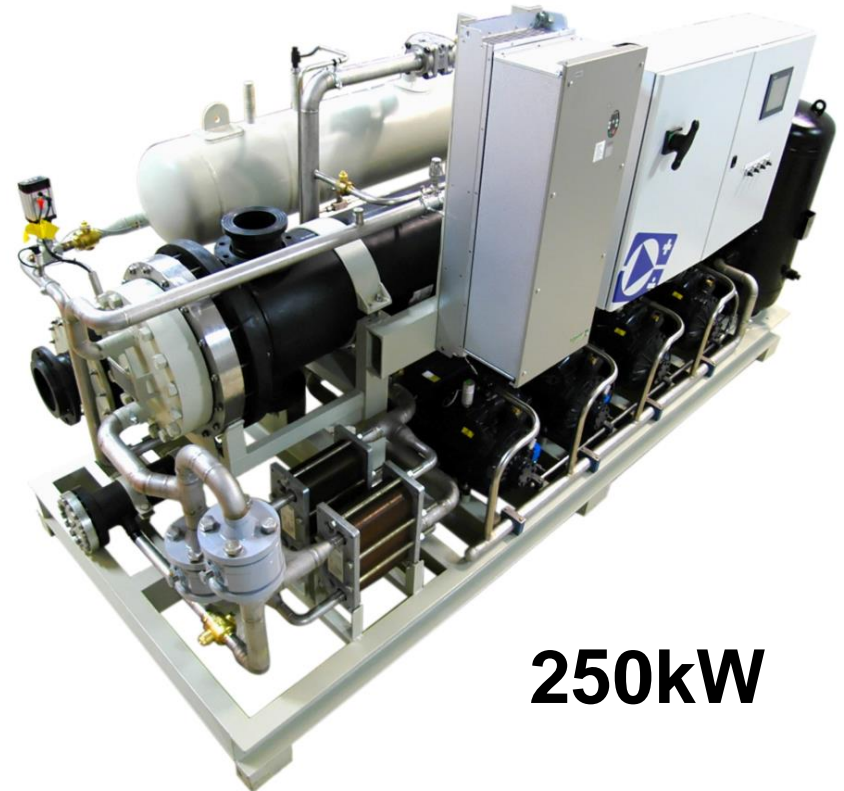
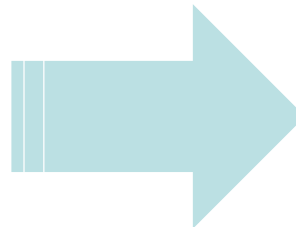
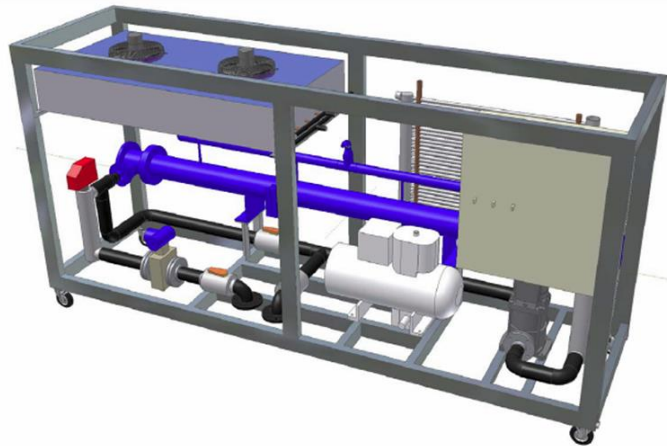
**Engineering, building
service**

- Motivation: introduction of CO₂ technology in a new market
 - Non-flammable, non-toxic replacement for R22
 - Environmental technology
 - Compact and efficient technology with good part-load characteristics
 - Primary market: retrofit for existing vessels in the coast fisheries

From laboratory to full scale commercial plant



40kW



250kW

First throw: 300 tonnes capelin!

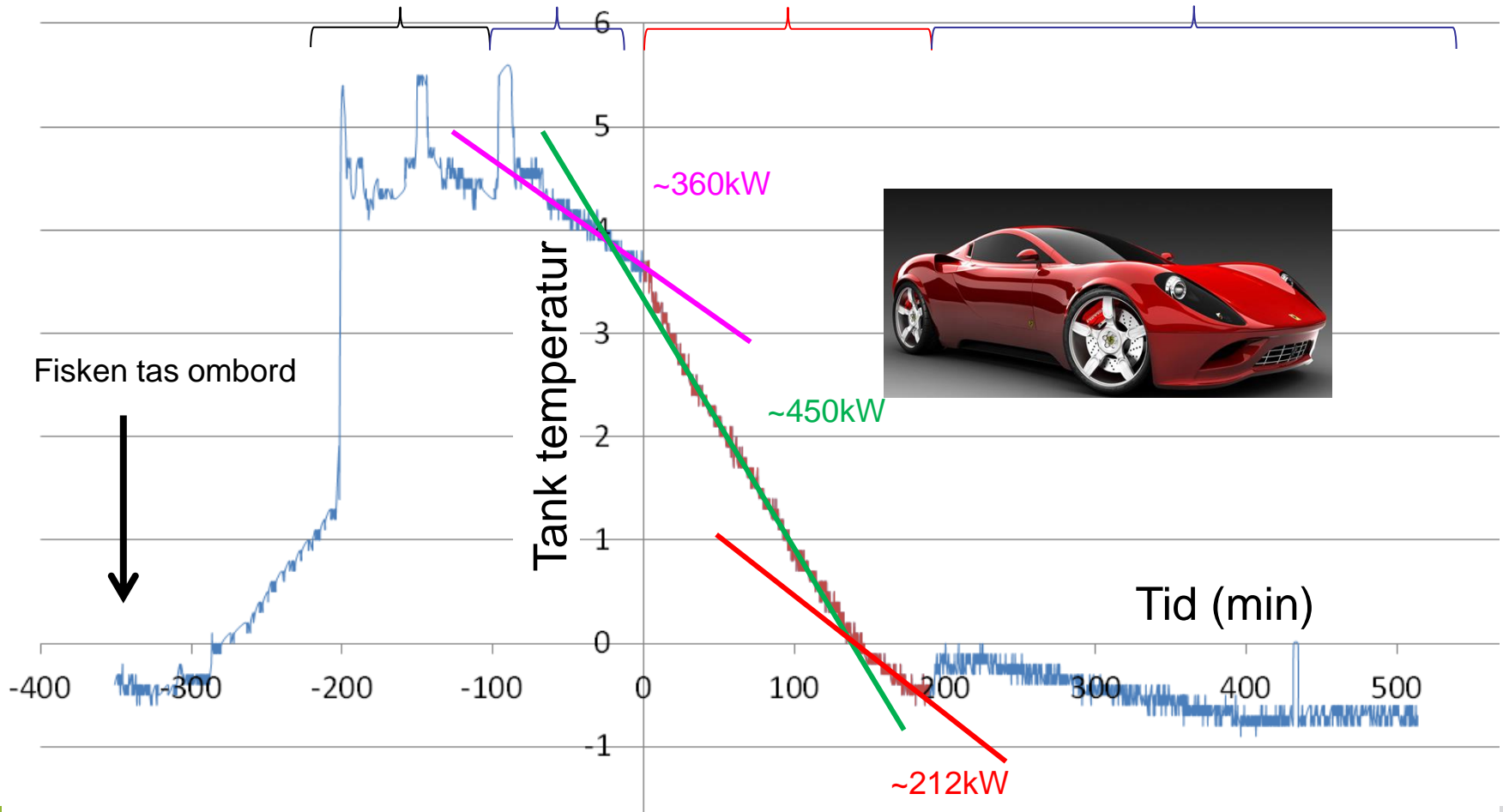


Utfordring
med
sirkulasjon

Last
480m³

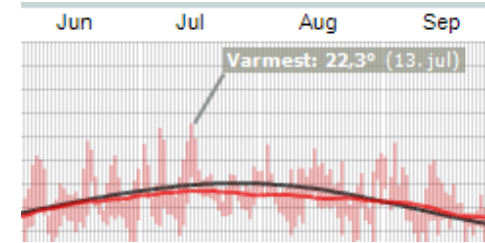
Last
284m³

Last
480m³



Summary first year of operation (cold!)

- Robust operation (tested in strong gale)
- Capacity control 100 -> 12.5% capacity, smooth start-up
- Approximately design capacity: 250kW 0 ->-1°C, -5°C evaporation temp
- High efficiency: COP = 4 - 6



Conclusions



- CREATIV has developed a lot of results contributing to more efficient refrigeration and heat pumping systems
- An important contribution has been made for the fundament for further development of energy efficient concepts
- Let us hope it will not be a "hvileskjær" in further development

We need to
reduce



Thank you for your attention!

This publication forms a part of the SINTEF – NTNU CREATIV project, performed under the strategic Norwegian research program RENERGI (195182/S60). The CREATIV is financially supported by the Research Council of Norway and several industry partners